CLAIM AMENDMENTS

- drives], comprising a brake body (1) mounted to float on a fixed bolt (7), a caliper (2) for straddling the periphery of a brake disc (3) [on the periphery thereof], and a solenoid (14) [integrated in brake body (1)] to magnetically attract a spring-biased armature [dis] disc (4) against said brake body (1), said armature disc (4) and the axially opposite portion of said caliper (2) having friction linings (5, 6) thereon to engage the two faces of brake disc (3), characterized in that a dual-arm rocker lever (8) is [pivotally] mounted on said fixed bolt (7) or on a bracket (17) [or on any component equivalent thereto] on said fixed bolt, the rocker lever [having a tongue (15B)] being moveable in the direction of the length of said fixed bolt and having two tongues (15A and 15B), one tongue (15A) connected to the peripheral surface of said armature disc (4) and the other tongue (15B) connected to said caliper, said connections being such that, as the brake disengages, said rocker lever or bracket (8) converts [the path] movement of said armature disc (4) to an oppositely directed movement of caliper (2) so as to provide an essentially equal air gap (s/2) on either side of said brake disc (3).
- 2. (Currently amended) Caliper brake as in claim 1, characterized in that <u>said</u> rocker lever <u>or bracket</u> (8) comprises two mutually parallel sections (8A, 8B) <u>[tangential to both sides of fixed bolt (7)</u>; <u>said sections clinging to] having friction linings (9) thereon which frictionally engage said fixed bolt (7) at diametrically opposite points [through adhesive friction</u>

exerted by friction linings (9)] to so define a movable fulcrum (13) on said fixed bolt (7).

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Rocker lever 8 is a member shaped of spring-grade steel strip to have two opposite limbs 8A, 8B engaging both sides of fixed bolt 7, said limbs having friction linings 9 on their inner surfaces to maintain the frictional engagement of the rocker lever with fixed bolt, as shown in Fig. 4. The axial frictional force exerted on the guide bolt is determined by the amount of the resilient outward deflection and by the nature of the friction linings. The rocker lever has two oppositely disposed flexible tongues 15A, 15B, with one tongue 15A connected to armature disc 4 and the other tongue 15B connected to caliper 2 both in a clearance-free manner. The lever ratio of the tongues to the center of the guide bolts is approximately 1:1. The friction linings 9 on the inner surfaces of the opposite limbs 8A, 8B of the rocker lever 8 tangentially contact diametrically opposite points on the fixed bolt 7 thereby frictionally pinching the bolt between the limbs at a fulcrum 13. Since the rocker lever 8 is frictionally movable along the fixed bolt 7, the fulcrum 13 is movable along the fixed bolt 7 so that the rocker lever may pivot about the moveable fulcrum 13 at different locations along the fixed bolt 7.

With the brake engaged and the position of the brake disc altered by some load change, tongues 15A, 15B act to axially shift the clamping mechanism in its entirety on the guide bolt. Energizing brake solenoid 14 will move armature disc 4 toward solenoid support 1. Armature-side tongue 15A transfers this movement to rocker lever 8, which will initiate an opposite directed movement of tongue 15B if its frictional engagement of guide bolt 7 is sufficient.

As a result, the air gaps between brake disc 3 and the two friction linings will be approximately the same and equal to s/2.

Fig. 5a shows another embodiment, the difference being that rocker lever 8 and its friction linings 9 cannot shift on bolt 7 directly; instead, another bracket 17 is provided and connected with bolt 7.

Figs. 5b, 5c show still another embodiment in which again a separate bracket 17 is connected with bolt 7, with axial displacement effected between rocker lever 8 and bracket 17 by the frictional engagement of friction lining 9.

Specification Replacement pages 4 and 4.1 (Marked up)

Rocker lever 8 is a member shaped of spring-grade steel strip to have two opposite limbs 8A, 8B engaging both sides of fixed bolt 7, said limbs having friction linings 9 on their inner surfaces to maintain the frictional engagement of the rocker lever with fixed bolt, as shown in Fig. 4. The axial frictional force exerted on the guide bolt is determined by the amount of the resilient outward deflection and by the nature of the friction linings. The rocker lever has two oppositely disposed flexible tongues 15A, 15B, with one tongue 15A connected to armature disc 4 and the other tongue 15B connected to caliper 2 both in a clearance-free manner. The lever ratio of the tongues to the center of the guide bolts is approximately 1:1. The friction linings 9 on the inner surfaces of the opposite limbs 8A, 8B of the rocker lever 8 tangentially contact diametrically opposite points on the fixed bolt 7 thereby frictionally pinching the bolt between the limbs at a fulcrum 13. Since the rocker lever 8 is frictionally movable along the fixed bolt 7, the fulcrum 13 is movable along the fixed bolt 7 so that the rocker lever may pivot about the moveable fulcrum 13 at different locations along the fixed bolt 7.

With the brake engaged and the position of the brake disc altered by some load change, tongues 15A, 15B act to axially shift the clamping mechanism in its entirety on the guide bolt. Energizing brake solenoid 14 will move armature disc 4 toward solenoid support 1. Armature-side tongue 15A transfers this movement to rocker lever 8, which will initiate an opposite directed movement of tongue 15B if its frictional engagement of guide bolt 7 is sufficient.

As a result, the air gaps between brake disc 3 and the two friction linings will be approximately the same and equal to s/2.

Fig. 5a shows another embodiment, the difference being that rocker lever 8 and its friction linings 9 cannot shift on bolt 7 directly; instead, another bracket 17 is provided and connected with bolt 7.

Figs. 5b, 5c show still another embodiment in which again a separate bracket 17 is connected with bolt 7, with axial displacement effected between rocker lever 8 and bracket 17 by the frictional engagement of friction lining 9.